

# Computer Controlled Systems

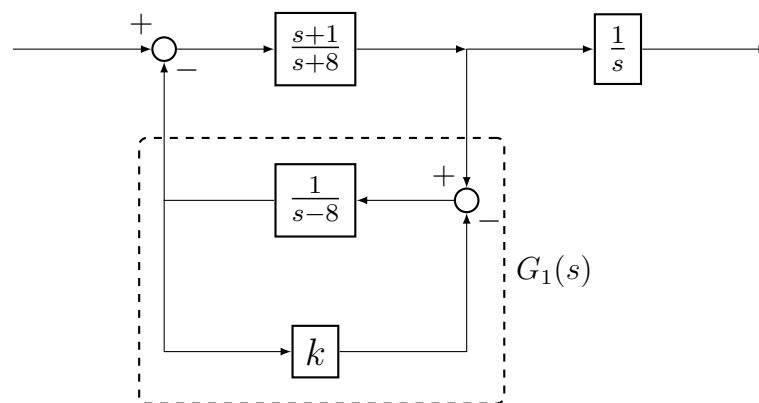
## Homework 3.

Submission deadline: 30th of November midnight via Moodle

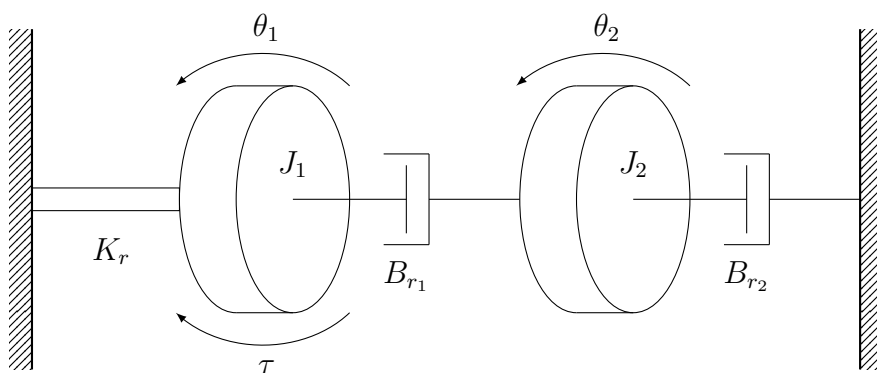
*All solutions are expected to be calculated by hand. The submission can be handwritten or typed in in LaTeX. Computer programs (e.g., Matlab) can be used for self-verification, but all problems have to contain the detailed steps of solutions.*

## Problems

- Let us consider the following block diagram:



- Compute the resulting transfer function  $G(s)$ . It is advised to first determine the auxiliary transfer function  $G_1(s)$  of the highlighted subsystem.
  - Choose the value of  $k$  such that resulting transfer function is stable.
- Let us consider the following rotating system with two flywheels:



The first flywheel has moment of inertia  $J_1$  and is attached by a flexible shaft with spring constant  $K_r$  to the wall. We apply an external torque  $\tau$  (which is the input). The second flywheel has moment of inertia  $J_2$  is driven by friction between the two flywheels with friction coefficient  $B_{r1}$  (which will be modeled with a damper). The second flywheel also has friction to the other wall with friction coefficient  $B_{r2}$ . The output of the system is the first flywheel's angle,  $\theta_1$ .

Using D'Alembert's Law we know that the sum of all torques (including the initial torque) should be zero; that is, for the first flywheel we can write the following equation:

$$\tau + J_1 \ddot{\theta}_1 + K_r \theta_1 + B_{r1} (\dot{\theta}_1 - \dot{\theta}_2) = 0.$$

Similarly for the second flywheel:

$$J_2\ddot{\theta}_2 + B_{r_2}\dot{\theta}_2 - B_{r_1}(\dot{\theta}_1 - \dot{\theta}_2) = 0.$$

The above equations can be rearranged in the form:

$$\begin{aligned}\ddot{\theta}_1 &= \frac{1}{J_1}(-\tau - K_r\theta_1 - B_{r_1}\dot{\theta}_1 + B_{r_1}\dot{\theta}_2), \\ \ddot{\theta}_2 &= \frac{1}{J_2}(-(B_{r_1} + B_{r_2})\dot{\theta}_2 + B_{r_1}\dot{\theta}_1).\end{aligned}$$

Using the state variables, input and output functions

$$x_1 = \theta_1,$$

$$x_2 = \dot{\theta}_1,$$

$$x_3 = \dot{\theta}_2,$$

$$u = \tau,$$

$$y = x_1$$

we can rewrite the equations describing the dynamics of the system as

$$\begin{aligned}\dot{x}_1 &= x_2, \\ \dot{x}_2 &= \frac{1}{J_1}(-u - K_r x_1 - B_{r_1} x_2 + B_{r_1} x_3), \\ \dot{x}_3 &= \frac{1}{J_2}(-(B_{r_1} + B_{r_2}) x_3 + B_{r_1} x_2), \\ y &= x_1\end{aligned}$$

and in matrix form

$$\begin{aligned}\dot{x} &= \begin{bmatrix} 0 & 1 & 0 \\ -\frac{K_r}{J_1} & -\frac{B_{r_1}}{J_1} & \frac{B_{r_1}}{J_1} \\ 0 & \frac{B_{r_1}}{J_2} & -\frac{B_{r_1}+B_{r_2}}{J_2} \end{bmatrix} x + \begin{bmatrix} 0 \\ -\frac{1}{J_1} \\ 0 \end{bmatrix} u; \\ y &= [1 \ 0 \ 0] x.\end{aligned}$$

(a) Substitute the following numbers and rewrite the state space representation:

$$K_r = 1 \quad B_{r_1} = 1 \quad B_{r_2} = 4 \quad J_1 = 0.2 \quad J_2 = 2.$$

(b) Is the system exponentially stable?

(c) Explain why the impulse response of the system is oscillating for a while!

(d) Design a static state feedback controller such that the closed loop system will not oscillate at all.

(e) Design an observer gain  $L$  such that the error dynamics of the Luenberger observer tends exponentially to zero.

**Compulsory only for TP students, *but extra points for others.***

(f) The energy stored in a flexible shaft (rotational spring) and a rotating mass is

$$\begin{aligned}E_K &= \frac{1}{2}K\theta^2 \\ E_J &= \frac{1}{2}J\dot{\theta}^2.\end{aligned}$$

Check that the total energy function  $V = E_{K_r} + E_{J_1} + E_{J_2}$  is an appropriate Lyapunov function.